# Lecture 8

# **Introduction to Contact Mechanics**

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## How to Model the Repulsive Interaction at Contact?

Atom-Atom?

Sphere-Plane?



Source: Capella & Dietler

Maybe if the contact area involves tens or hundreds of atoms the description of net repulsive force is best captured by continuum elasticity models



### What we want to know

Nature of the contact - reversible (elastic)? hysteretic?

Contact radius (contact area) as function of applied force

Any deformation?

Pull-off force (adhesion force)

What determines all these quantities?



### Continuum description of contact - history

- Hertz (1881) takes into account neither surface forces nor adhesion, and assumes a linearly elastic sphere indenting an elastic surface
- Sneddon's analysis (1965) considers a rigid sphere (or other rigid shapes) on a linearly elastic half-space.
- Neither Hertz or Sneddon consider surface forces discussed in last lecture.
- Bradley's analysis (1932) considers two <u>rigid</u> spheres interacting *via* the Lennard-Jones 6-12 potential
- Derjaguin-Müller-Toporov (DMT, 1975) considers an elastic sphere with rigid surface but includes van der Waals forces outside the contact region. Applicable to stiff samples with low adhesion.
- Johnson-Kendall-Roberts (JKR, 1971) neglects long-range interactions o utside contact area but includes short-range forces in the contact area. Applicable to soft samples with high adhesion.
- Maugis (1992) theory is even more accurate shows that JKR and DMT are limits of same theory



### **Tip-sample Interaction Models**



Rigid tip-rigid sample

Deformable tip and rigid sample\*

From the Derjaguin approximation for <u>rigid tip</u> interacting with <u>rigid</u> <u>sample</u> we have

 $F_{tip-sample}(r^*) = F_{adhesion} = 2\pi R_{tip} U(r^*) \simeq 2\pi R_{tip} W_{132} = 2\pi R_{tip} (\gamma_{13} + \gamma_{23} - \gamma_{12})$ 

- Real tips and samples are not rigid. Several theories are used to better account for this fact (Hertz, DMT, JKR)
- \* These theories also apply to elastic samples, they are just shown on rigid sample to demonstrate key quantities clearly. For example D is the combined tip-sample deformation in (b)

## I. Surface energies - notation

- Work of adhesion and cohesion: work done to separate unit areas of two media 1 and 2 from contact to infinity in vacuum. If 1 and 2 are different then  $W_{12}$  is the work of adhesion; if 1 and 2 are the same then  $W_{11}$  is the work of cohesion. Think vdW's whenever you see work of adhesion/cohesion.
- Surface energy: This is the free energy change  $\gamma$  when the surface area of a medium is increased by unit area. Thus  $W_{11} = 2\gamma_1$
- While separating dissimilar materials the free energy change in producing an "interfacial" area by unit area is known as their interfacial energy  $W = \chi + \chi = \chi$

$$W_{12} = \gamma_1 + \gamma_2 - \gamma_{12}$$

Work of adhesion in a third medium





### II. What is the "Stiffness" of the Tip/Substrate?





### Standard results



#### **Deformation D:**

$$D_{Hertz} = \frac{a^{2}}{R_{tip}} = \left(\frac{F^{2}}{R_{tip}E_{tot}^{2}}\right)^{1/3} \qquad D_{DMT} = \frac{a^{2}}{R_{tip}} = \left(\frac{\left(F + 2\pi R_{tip}W_{132}\right)^{2}}{R_{tip}E_{tot}^{2}}\right)^{1/3}$$
$$D_{JKR} = \frac{a^{2}}{R_{tip}} - \frac{2}{3}\sqrt{\frac{6\pi W_{132}a}{E_{tot}}} \qquad \qquad \frac{1}{E_{tot}} = \frac{3}{4}\left(\frac{1 - v_{s}^{2}}{E_{s}} + \frac{1 - v_{t}^{2}}{E_{t}}\right) = \frac{3}{4}\frac{1}{E^{*}}$$
$$Sometimes \quad E_{tot} = \frac{4}{3}E^{*}$$

- 1/3

### Standard results (cont.)

Pull-off Force F:



$$F_{adhesion}^{Hertz} = 0$$

$$F_{adhesion}^{DMT} = 2\pi R_{tip} W_{132}$$

$$F_{adhesion}^{JKR} = \frac{3\pi}{2} \mathbf{R}_{tip} \mathbf{W}_{132}$$



Source: Butt, Cappella, Kappl

### Example

<u>Hertz contact</u>:  $R_{tip} = 30 \text{ nm}; F_{app} = 1 \text{ nN}$   $E_{tip} = E_{sub} = 200 \text{ Gpa}; \text{ Poisson ratio} = v_{tip} = v_{sub} = 0.3 = v$   $\frac{1}{E_{tot}} = \frac{3}{4} \left( \frac{1 - v_{sub}^2}{E_{sub}^2} + \frac{1 - v_{tip}^2}{E_{tip}^2} \right) = \frac{3}{2} \frac{1 - v^2}{E}$  $= \frac{3}{2} \frac{0.91}{(200 \text{ GPa})} = \frac{1.365}{200 \text{ GPa}} \Rightarrow E_{tot} = 146.5 \text{ GPa}$ 



#### Contact radius:

$$a_{Hertz} = \left(\frac{R_{tip}F}{E_{tot}}\right)^{\frac{1}{3}} = 0.59 \ nm$$

**Deformation:** 

$$D_{Hertz} = \frac{a^2}{R_{tip}} = \left(\frac{F^2}{R_{tip}E_{tot}^2}\right)^{1/3} = 12 \ pm \qquad \text{Pull-off Force=0}$$
Contact Pressure:  $P \simeq \frac{F}{\pi a_{Hertz}^2} = 0.9 \ GPa \approx 9000 \ atmos.$ 
RDUE











$$F_{ts}(d) = \begin{cases} -\frac{HR}{6d^2}, & d > a_o \\ -\frac{HR}{6a_o^2} + \frac{4}{3}E^*\sqrt{R}(a_o - d)^{\frac{3}{2}}, & d \le a_o \end{cases}$$

$$E^{*} = \left[\frac{1 - v_{tip}^{2}}{E_{tip}} + \frac{1 - v_{sample}^{2}}{E_{sample}}\right]^{-1}$$







### **Contact forces: Maugis' Theory**



### Validity of different models – converting measured adhesion force to work of adhesion



### Comments on these theories

- JKR predicts infinite stress at edge of contact circle.
- In the limit of small adhesion JKR -> DMT
- Most equations of JKR and Hertz and DMT have been tested experimentally on molecularly smooth surfaces and found to apply extremely well
- Most practical limitation for AFM is that no tip is a perfect smooth sphere, small asperities make a big difference.
- Hertz, DMT describe conservative interaction forces, but in JKR, the interaction itself is non-conservative (why?) ...for a force to be considered conservative it has to be describable as a gradient of potential energy.



### Combining van der Waals force & DMT contact

$$F_i(z) = \begin{cases} -\frac{AR}{6z^2}, & (\text{for } z > a_0) \\ -\frac{AR}{6a_0^2} + \frac{4}{3}E^*\sqrt{R}(a_0 - z)^{3/2}, & (\text{for } z \le a_0) \end{cases}$$

